

CLAIM AMENDMENTS

1 1. (currently amended) A drive for reciprocating a roll
2 stand of a pilger cold-rolling system, the drive comprising:

3 a crank shaft defining an axis transverse to a
4 reciprocation direction of the stand;

5 a crank-shaft gear fixed on the crank shaft;

6 a crank on the crank shaft and rotatable therewith about
7 ~~[[an]] the axis transverse to a reciprocation direction of the~~
8 ~~stand;~~

9 a tie rod having an outer end journaled on the stand and
10 an inner end eccentrically journaled on the crank, whereby rotation
11 of the crank shaft reciprocates the stand in the direction;

12 a compensating weight fixed eccentrically to the crank
13 opposite the tie rod and orbiting in a weight plane substantially
14 perpendicular to the axis on rotation of the crank;

15 a single counterweight shaft offset along the plane from
16 the crank shaft and substantially parallel thereto;

17 a counterweight gear fixed on the counterweight shaft;

18 a single counterweight fixed eccentrically on the
19 counterweight shaft and offset along the weight plane from the
20 crank, one of the weights being integrally formed with the
21 respective gear; and

22 drive means connected to the counterweight shaft for
23 rotating same and ~~connecting the counterweight to the crank for~~

24 thereby synchronously orbiting the counterweight and the
25 compensating weight in the weight plane on rotation of the crank.

2. (canceled)

1 3. (currently amended) The pilger roll-stand drive
2 defined in claim [[2]] 1 wherein the drive means further comprises
3 a drive shaft carrying a gear meshing with the gear of
4 the counterweight shaft ~~carrying the counterweight.~~

1 4. (original) The pilger roll-stand drive defined in
2 claim 3 wherein the shafts are rotatable about axes that are all
3 coplanar and parallel.

1 5. (original) The pilger roll-stand drive defined in
2 claim 4 wherein the stand is centered on the weight plane.

1 6. (original) The pilger roll-stand drive defined in
2 claim 4 wherein the crank is centered on the weight plane.

1 7. (original) The pilger roll-stand drive defined in
2 claim 4, further comprising
3 a second crank coaxial with and connected to the first-
4 mentioned crank;
5 a second tie rod having an outer end journaled on the
6 stand and an inner end eccentrically journaled on the second crank;

7 a second compensating weight fixed to the second crank
8 opposite the tie rod and orbiting in a second weight plane parallel
9 to but offset from the first-mentioned weight plane on rotation of
10 the cranks; and

11 a second counterweight offset along the second weight
12 plane from the second crank, the drive means also connecting the
13 second counterweight to the second crank for orbiting the second
14 counterweight in the second weight plane on rotation of the second
15 crank.

1 8. (original) The pilger roll-stand drive defined in
2 claim 7 wherein the roll stand is centered on a plane symmetrically
3 flanked by the first and second weight planes.

1 9. (original) The pilger roll-stand drive defined in
2 claim 7 wherein the drive means is offset from the planes.

1 10. (currently amended) The pilger roll-stand drive
2 defined in claim 4 wherein both of the gears are unitarily formed
3 with the respective weights.

1 11. (original) The pilger roll-stand drive defined in
2 claim 4 wherein the shafts are horizontal.

1 12. (original) The pilger roll-stand drive defined in
2 claim 4 wherein the shafts are vertical.

1 13. (original) The pilger roll-stand drive defined in
2 claim 4 wherein the crank has a pin on which the inner end of the
3 tie rod is journaled and that is formed with passages through which
4 a lubricant can be fed.

1 14. (new) A drive for reciprocating a roll stand of a
2 pilger cold-rolling system, the drive comprising:

3 a crank shaft defining an axis transverse to a
4 reciprocation direction of the stand;

5 a crank-shaft gear fixed on the crank shaft;

6 a crank on the crank shaft and rotatable therewith about
7 the axis;

8 a tie rod having an outer end journaled on the stand and
9 an inner end eccentrically journaled on the crank, whereby rotation
10 of the crank shaft reciprocates the stand in the direction;

11 a compensating weight unitarily formed with the crank-
12 shaft gear and fixed eccentrically to the crank opposite the tie
13 rod and orbiting in a weight plane substantially perpendicular to
14 the axis on rotation of the crank;

15 a single counterweight shaft offset along the plane from
16 the crank shaft and substantially parallel thereto;

17 a counterweight gear fixed on the counterweight shaft;

18 a single counterweight unitarily formed with the
19 counterweight gear fixed eccentrically on the counterweight shaft
20 and offset along the weight plane from the crank; and
21 drive means connected to the counterweight shaft for
22 rotating same and for thereby synchronously orbiting the
23 counterweight and the compensating weight in the weight plane.